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A thin plate approximation for ocean wave interactions with an ice shelf

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A variational principle is proposed to derive the governing equations for the problem of 8 ocean wave interactions with a floating ice shelf, where the ice shelf is modelled by the full 9 linear equations of elasticity and has an Archimedean draught. The variational principle is 10 used to form a thin-plate approximation for the ice shelf, which includes water-ice coupling 11 at the shelf front and extensional waves in the shelf, in contrast to the benchmark thin-plate 12 approximation for ocean wave interactions with an ice shelf. The thin-plate approximation 13 is combined with a single-mode approximation in the water, where the vertical motion is 14 constrained to the eigenfunction that supports propagating waves. The new terms in the 15 approximation are shown to have a major impact on predictions of ice shelf strains for wave 16 periods in the swell regime. 17

18 Key words: N/A

19 1. Introduction

20 Flexural waves are known to propagate through floating ice from classical experimental studies (e.g., Press et al. 1951), and it is known from observations that the flexure can be 21 forced by ocean waves (e.g., Holdsworth 1969). For over half a century, thin elastic plates 22 (Lamb 1916) floating on water have been the benchmark model for ocean wave-induced 23 flexural motions of sea ice (Evans & Davies 1968; Wadhams et al. 1988; Meylan & Squire 24 1994; Vaughan et al. 2009; Montiel et al. 2016; Pitt et al. 2022) and ice shelves (Holdsworth & 25 Glynn 1978; Vinogradov & Holdsworth 1985; Fox & Squire 1991b; Williams & Squire 2007; 26 Papathanasiou et al. 2015; Meylan et al. 2021). The benchmark model, which dates back to 27 Greenhill (1916), assumes the vertical ice displacements are uniform with respect to thickness 28 (i.e., a thin plate), and the water is a potential-flow fluid. The plate appears in the model 29 through flexural and inertial restoring forces at the water surface, which are manifested as 30 high-order derivatives in the dynamic surface condition. The high-order boundary condition 31 supports so-called flexural-gravity waves, plus wave modes that have no analogue in open 32

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water (i.e., where the water surface is in contact with air), which are typically oscillatory decaying waves but can become purely decaying in certain regimes (Williams 2006; Bennetts
 2007), as well as evanescent (exponentially decaying) modes.

In both sea ice and ice shelf applications, the canonical wave-ice interaction problem 36 involves a two-dimensional water domain (one horizontal dimension plus depth), which has 37 half of its surface covered by ice, and where motions are excited by an incident wave from 38 39 the open (non-ice covered) water (Evans & Davies 1968; Tkacheva 2001; Linton & Chung 2003). The incident wave is partially reflected at the ice edge and partially transmitted into 40 the ice-covered domain. The model is used to predict, e.g., strains in landfast sea ice (Fox 41 & Squire 1991b, 1994) and ice shelves (Fox & Squire 1991a), and is the basis for models 42 of wave attenuation in the marginal ice zone (Bennetts & Squire 2012b,a). Although the 43 Archimedean draught of ice is $\approx 90\%$ of its thickness, the thinness of sea ice has been used 44 to justify the so-called shallow-draught approximation, in which the ice floats at the water 45 surface with no submergence. Therefore, the ice edge experiences no loading, and free edge 46 conditions are applied (i.e., zero bending moment and shear stress). The water and ice are 47 coupled along the underside of the ice only. 48

Methods have been developed to accommodate Archimedean ice draught, whilst retaining 49 the free edge conditions (Williams & Porter 2009; Montiel et al. 2012; Papathanasiou et al. 50 2019). The methods address the geometrical corner created by the partial submergence of the 51 ice edge, but not the additional water-ice coupling created by the bending moment applied 52 by the water motion on the ice edge and the kinematic coupling between the ice edge and the 53 water (equality of the normal water and ice displacements at their interface). Notably, Porter 54 & Porter (2004), and subsequently Bennetts et al. (2007), who corrected an error in Porter 55 & Porter (2004), derived the incorrect free edge conditions as the natural conditions of a 56 variational principle, but where the thinness of the plate was already applied in the underlying 57 Lagrangian, i.e., a one dimensional body was partially submerged in a two-dimensional fluid. 58 Although ad-hoc, the use of the shallow-draught approximation and/or free edge conditions 59 at a sea ice edge seems unlikely to have a major impact on model predictions, as the 60 ice thickness (typically tens of centimetres to a few metres) is much smaller than other 61 characteristic lengths. Relevant wavelengths are in the swell regime (tens to hundreds of 62 metres; wave periods 10-30 s) and wave-sea ice interactions typically occur in the deep 63 ocean (> 1 km, i.e., much greater than wavelengths). In contrast, ice shelves are hundreds of 64 65 metres thick, occur on continental shelves and the sub-ice shelf water cavities are typically hundreds of metres deep. Ice shelves vibrate in response to ocean waves from long swell 66 (wavelengths on the order of hundreds of metres) to infragravity waves (wavelengths on the 67 order of kilometres to tens of kilometres; wave periods 50-300 s) and longer (Chen et al. 68 2019). Therefore, the jump in water depth created by the ice draught affects model predictions 69 (Kalyanaraman et al. 2019). 70

For the ice shelf application, water-ice coupling at the submerged portion of the shelf 71 front (i.e., the ice edge) appears likely to influence model predictions for incident swell. 72 Compelling evidence that swell forced shelf front strains strong enough to trigger runaway 73 ice shelf disintegrations makes this missing aspect of the benchmark thin-plate model 74 conspicuous (Massom et al. 2018). Abrahams et al. (2023) recently analysed a numerical 75 time domain simulation, in which the ice shelf is modelled using the full (linear) equations 76 of elasticity. In addition to flexural waves, they identified extensional waves in the shelf that 77 are generated by water-ice coupling at the shelf front. There is also observational evidence 78 of ocean waves forcing extensional waves in ice shelves (Chen et al. 2018). (See Hunkins 79 1960, for observations of extensional waves in sea ice.) Further, Abrahams et al. (2023) 80 81 showed that extensional wave displacement amplitudes exceed those of the flexural waves for low frequencies, with the extensional to flexural amplitude ratio tending to infinity as 82

the frequency tends to zero. Kalyanaraman *et al.* (2020) analysed numerical computations in the frequency domain of an ice shelf (of finite length) modelled using the full equations of elasticity (although neglecting gravity), but applied free edge conditions at the shelf front. They found the flexural displacement profiles were similar to those predicted by the benchmark model, at least for two wave periods in the infragravity regime. The finding is broadly consistent with the results of studies using the shallow-draught approximation and thick plate models (Fox & Squire 1991*a*; Balmforth & Craster 1999).

In this article, we outline a variational principle that derives the governing equations of the 90 ice shelf problem, where the shelf has an Archimedean draught and is modelled by the full 91 equations of elasticity, i.e., no simplifying assumptions are made about the ice displacements. 92 We use the variational principle to derive a thin plate approximation by constraining the ice 93 displacements to low-order subspaces, with the underlying assumption that the ice thickness 94 is small with respect to the wavelengths it supports. The thin plate approximation extends 95 the benchmark model by including extensional waves in the shelf and coupling water and 96 ice motions at the shelf front. We combine the thin-plate approximation with a single-mode 97 approximation in the water, which involves averaging with respect to depth, similar to Porter 98 & Porter (2004) and Bennetts et al. (2007). We use the approximations to investigate the 99 influence of coupling at the ice edge and extensional waves on ice shelf strains, across the 100 swell and infragravity wave regimes. 101

102 **2. Preliminaries**

Consider a two-dimensional domain of homogeneous, inviscid and irrotational water, which has an (undisturbed) finite depth H and infinite horizontal extent (Fig. 1). An ice shelf of finite thickness h and semi-infinite length covers the surface of the right-hand side of the water domain. Let the Cartesian coordinate system $(x, z) \equiv (x_1, x_2)$ define locations in the water and ice shelf. The horizontal coordinate, $x \in \mathbb{R}$, has its origin set to coincide with the shelf front. The vertical coordinate, z, has its origin set to coincide with the undisturbed water surface, such that the (flat) bed is located at z = -H.

The ice shelf is assumed to be a homogenous, isotropic, purely elastic solid without gravitational pre-stress (see Appendix A for evidence the gravitational pre-stress has little effect on wave propagation). It has an Archimedean draught, such that its (undisturbed) lower surface is located at

$$z = -d \equiv -\frac{\rho_1 n}{\rho_w},\tag{2.1}$$

where $\rho_i = 922.5 \text{ kg m}^{-3}$ and $\rho_w = 1025 \text{ kg m}^{-3}$ are the ice and water densities, respectively, such that $\rho_i / \rho_w = 0.9$. The ice/water domain is partitioned into the ice shelf, the sub-shelf water cavity and the open ocean (Fig. 1), respectively,

118
$$\Omega_{\rm sh} = \{(x, z) : 0 < x < \infty; -d < z < h - d\}$$
(2.2*a*)

114

$$\Omega_{\rm ca} = \{ (x, z) : 0 < x < \infty; -H < z < -d \},$$
(2.2b)

129

and
$$\Omega_{op} = \{(x, z) : -\infty < x < 0; -H < z < 0\}.$$
 (2.2c)

The sub-domains (2.2) are assumed to be the equilibrium state of the ice/water system, about which motions are forced by incident waves.

Small amplitude (linear) motions of the ice–water system are considered. Let the displacement field be

126
$$\mathbf{u}(x,z,t) = [U(x,z,t); W(x,z,t)] \equiv [U_1(x,z,t); U_2(x,z,t)].$$
(2.3)



Figure 1: Schematic (not to scale) of the equilibrium geometry.

The displacement in the y-direction (or x_3 -direction; which points out of the page in Fig. 1) is V or $U_3 \equiv 0$. In the ice, the infinitesimal strain tensor, $\varepsilon(x, z, t)$, is defined as

129
$$\varepsilon_{ij} \equiv \frac{1}{2} \left(U_{j,x_i} + U_{i,x_j} \right) \quad \text{for} \quad i,j \in \{1,2,3\}.$$
(2.4)

130 The Cauchy stress tensor, $\sigma(x, z, t)$ (i.e., the stress tensor under infinitesimal deformation),

131 is related to the strain tensor via the standard constitutive relations, such that

132
$$\varepsilon_{ij} = -\frac{\nu}{E} \,\delta_{ij} \sum_{r=1}^{3} \sigma_{rr} + \frac{1+\nu}{E} \,\sigma_{ij} \quad \text{for} \quad i, j \in \{1, 2, 3\}, \tag{2.5}$$

where *E* is Young's modulus and *v* is Poisson's ratio, and E = 11 GPa and v = 0.3 are used as standard values for ice shelves. Plane strain is assumed in the *x*-*z* plane, i.e., $\varepsilon_{3i} = \varepsilon_{i3} = 0$ (for i = 1, 2, 3) but σ_{33} is non-zero.

136 In the water, which is modelled as inviscid, the stress tensor has components

137
$$\sigma_{ij} = -P \,\delta_{ij} \quad \text{for} \quad i, j \in \{1, 2, 3\},$$
 (2.6)

where P(x, z, t) is the pressure field. Assuming the water undergoes irrotational motions in the *x*-*z* plane (with no motion in the *y*-direction), the displacement field is expressed as the gradient of a scalar displacement potential, $\Phi(x, z, t)$. At this stage, no relation is assumed between the pressure and the displacement potential, i.e., the Bernoulli equation is not applied. The functions $\zeta_{\bullet}(x, t)$ denote the vertical displacements of the water–atmosphere, water–ice and ice–atmosphere interfaces (• = w–a, w–i, i–a, respectively). They are not yet related to the ice displacements (**u**), or water pressure (*P*) or displacements (through Φ).

The relative hydrostatic pressures in the open ocean, ice shelf and sub-shelf water cavity are, respectively,

$$P_{\rm op}(z) = -\rho_{\rm w} g z, \ P_{\rm sh}(z) = P_0 - \rho_{\rm i} g (z+d) \text{ and } P_{\rm ca}(z) = P_0 - \rho_{\rm w} g (z+d) = -\rho_{\rm w} g z,$$

(2.7a,b,c)

148 where

147

149

$$P_0 = P_{\rm sh}(-d) = P_{\rm ca}(-d) = P_{\rm op}(-d) = \rho_{\rm i} g h = \rho_{\rm w} g d, \qquad (2.8)$$

and $g = 9.81 \text{ m s}^{-2}$ is the constant gravitational acceleration. Note that $P_{\rm sh}(h-d) = P_{\rm op}(0) = 0$, so Eq. (2.7a–c) represent the true hydrostatic pressure minus the constant atmospheric pressure, $P_{\rm at}$, and that the hydrostatic pressure is continuous going from the open ocean into the sub-shelf equity.

153 the sub-shelf cavity.

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(3.5)

3. Variational principle 154

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3.1. Lagrangian

The Lagrangian for the ice-water system is 156

157
$$\mathcal{L}\{\mathbf{u}, \Phi, P, \zeta, \tau\} = \mathcal{L}_{sh}\{\mathbf{u}, \zeta_{i-a}, \zeta_{w-i}, \tau\} + \mathcal{L}_{ca}\{\Phi, P, \zeta_{w-i}, \tau\} + \mathcal{L}_{op}\{\Phi, P, \zeta_{w-a}, \tau\}, \quad (3.1)$$

where \mathcal{L}_{sh} , \mathcal{L}_{ca} and \mathcal{L}_{op} are the Lagrangians for the ice shelf, sub-shelf water cavity and 158 open ocean, respectively. 159

The (linearised) Lagrangian for the ice shelf is expressed as $\mathcal{L}_{sh} = \mathcal{T}_{sh} - \mathcal{V}_{sh}$, where \mathcal{T}_{sh} and 160 \mathcal{V}_{sh} are the kinetic and potential energies in the ice shelf, respectively. The kinetic energy is 161

162
163
$$\mathcal{T}_{\rm sh}\{\mathbf{u}\} = \frac{\rho_{\rm i}}{2} \iint_{\Omega_{\rm sh}} \left\{ U_t^2 + W_t^2 \right\} \, \mathrm{d}x \, \mathrm{d}z. \tag{3.2}$$

The potential energy is the integral of the strain energy density plus the gravitational potential 164 over the shelf domain, plus integrals from linearisation of the moving boundaries and normal 165 stresses applied to the boundaries (denoted τ_{ii} ; applied shear stress, τ_{ii} for $i \neq j$, are neglected 166 as the surrounding water and air do not support them). The strain energy density is 167

168
169
$$v_{e}(\varepsilon) = \frac{1}{2} \sum_{i=1}^{2} \sum_{j=1}^{2} \sigma_{ij} \varepsilon_{ij},$$
(3.3)

which depends only on the strain since (2.5) can be inverted to write the stress in terms of 170 the strain. 171

172 The gravitational potential is calculated relative to the upper surface of the shelf (z = h - d), 173 as

173
$$P_{\rm sh}(z-W) = \rho_{\rm i} g (W-z+h-d). \tag{3.4}$$

Therefore, the potential energy in the ice shelf is 176

177
$$\mathcal{V}_{\rm sh}\{\mathbf{u},\zeta_{\rm i-a},\zeta_{\rm w-i},\boldsymbol{\tau}\} = \iint_{\Omega_{\rm sh}} \left(\frac{1}{2}\sum_{i=1}^{2}\sum_{j=1}^{2}\sigma_{ij}\,\varepsilon_{ij} + \rho_{\rm i}\,g\left(W - z + h - d\right)\right) \mathrm{d}x\,\mathrm{d}z$$

178
$$+ \int_0^\infty \left[\rho_i g W \zeta_{i-a} - \frac{1}{2} \rho_i g \zeta_{i-a}^2 - \tau_{22} W \right]_{z=h-d} dx$$

179
$$-\int_0^\infty \left[(P_0 + \rho_i g W) \zeta_{w-i} - \frac{1}{2} \rho_i g \zeta_{w-i}^2 - \tau_{22} W \right]_{z=-d} dx$$

180 +
$$\int_{-d}^{h-d} \left[\tau_{11} U \right]_{x=0} dz$$

 $-\int_{-d}^{h-d} \left[\tau_{11} U\right]_{x\to\infty} \mathrm{d}z.$ 181 182

The atmospheric pressure, $P_{\rm at}$, appears implicitly in (3.5) via the applied stresses 183

184
$$[\tau_{11}]_{x=0,00,z=h-d} \equiv -P_{\text{at}}.$$
 (3.6)

The Lagrangian for the sub-shelf water cavity is expressed as $\mathcal{L}_{ca} = \mathcal{T}_{ca} - \mathcal{V}_{ca}$, where the 185

$$P_{\rm sh}(z-W) = \rho_{\rm i} g (W - z + h - d). \tag{3.4}$$

7
$$\mathcal{V}_{\rm sh}\{\mathbf{u},\zeta_{\rm i-a},\zeta_{\rm w-i},\boldsymbol{\tau}\} = \iint_{\Omega_{\rm sh}} \left(\frac{1}{2}\sum_{i=1}^{2}\sum_{j=1}^{2}\sigma_{ij}\varepsilon_{ij}+\rho_{\rm i}g\left(W-z+h-d\right)\right)dx\,dz$$

186 kinetic energy in the water cavity is

187
188

$$\mathcal{T}_{ca}\{\Phi\} = \frac{\rho_{w}}{2} \iint_{\Omega_{ca}} \left\{\Phi_{xt}^{2} + \Phi_{zt}^{2}\right\} dx dz.$$
(3.7)

189 For the potential energy, a term that is analogous to the strain energy density in the ice is

$$v_{\rm e}(P,\Phi) = -P\,\nabla^2\Phi,\tag{3.8}$$

and the gravitational potential is relative to the water surface (without the ice shelf; z = 0), i.e.,

$$P_{\rm ca}(z-\Phi_z)=\rho_{\rm i}\,g\,(\Phi_z-z). \tag{3.9}$$

196 Therefore, the potential energy is

197
$$\mathcal{V}_{ca}\{\Phi, P, \zeta_{w-i}, \tau\} = \iint_{\Omega_{ca}} \left\{ -P \nabla^2 \Phi + \rho_i g \left(\Phi_z - z \right) \right\} dx dz$$

198
$$+ \int_0^\infty \left[(P_0 + \rho_w g \Phi_z) \zeta_{w-i} - \frac{1}{2} \rho_w g \zeta_{w-i}^2 - \tau_{22} \Phi_z \right]_{z=-d} dx$$

199
$$-\int_0^\infty \left[-\tau_{22}\,\Phi_z\right]_{z=-H}\,\mathrm{d}x$$

200
201
$$-\int_{-H}^{-d} \left[\tau_{11} \Phi_x\right]_{x=0}^{\infty} dz.$$
 (3.10)

Similarly, the linearised Lagrangian for the open ocean is $\mathcal{L}_{op} = \mathcal{T}_{op} - \mathcal{V}_{op}$, in which

203
204
205

$$\mathcal{T}_{op}\{\Phi\} = \frac{\rho_{w}}{2} \iint_{\Omega_{op}} \left\{\Phi_{xt}^{2} + \Phi_{zt}^{2}\right\} dx dz$$
(3.11)

and
$$\mathcal{V}_{op}\{\Phi, P, \zeta_{w-a}, \tau\} = \iint_{\Omega_{op}} \{-P \nabla^2 \Phi + \rho_i g (\Phi_z - z)\} dx dz$$

207
$$+ \int_{-\infty}^{0} \left[\rho_{w} g \Phi_{z} \zeta_{w-a} - \frac{1}{2} \rho_{w} g \zeta_{w-a}^{2} - \tau_{22} \Phi_{z} \right]_{z=0} dx$$

$$-\int_{-\infty}^{0} \left[-\tau_{22} \Phi_z \right]_{z=-H} \mathrm{d}x$$

209
210 +
$$\int_{-H}^{0} \left[-\tau_{11} \Phi_x \right]_{x \to -\infty}^{0} dz.$$
 (3.12)

211 Again, the atmospheric pressure appears implicitly, via

212
$$[\tau_{22}]_{x<0,z=0} \equiv -P_{at}.$$
 (3.13)

Small variations are applied to all unknowns, such that the Lagrangians become

214
$$\mathcal{T}_{sh}\{\mathbf{u} + \delta \mathbf{u}\} = \mathcal{T}_{sh}\{\mathbf{u}\} + \delta \mathcal{T}_{sh}\{\mathbf{u} : \delta \mathbf{u}\} + o(\delta \mathbf{u}), \text{ and so on.}$$
(3.14)

215 The first variation of the full Lagrangian, $\delta \mathcal{L}\{\mathbf{u}, \Phi, P, \zeta, \tau : \delta \mathbf{u}, \delta \Phi, \delta P, \delta \zeta, \delta \tau\}$, is

216
$$\delta \mathcal{L} = \delta \mathcal{L}_{sh} + \delta \mathcal{L}_{ca} + \delta \mathcal{L}_{op} = \delta \mathcal{T}_{sh} - \delta \mathcal{V}_{sh} + \delta \mathcal{T}_{ca} - \delta \mathcal{V}_{ca} + \delta \mathcal{T}_{op} - \delta \mathcal{V}_{op}.$$
(3.15)

3.2. Action

The action, \mathcal{A} , is the integral of the Lagrangian over an arbitrary time interval, $t_0 < t < t_1$, i.e.,

$$\mathcal{A}\{\mathbf{u}, \Phi, P, \zeta, \tau\} = \int_{t_0}^{t_1} \mathcal{L}\{\mathbf{u}, \Phi, P, \zeta, \tau\} \,\mathrm{d}t.$$
(3.16)

221 Its first variation is

$$\delta \mathcal{A}\{\mathbf{u}, \Phi, P, \zeta, \tau : \delta \mathbf{u}, \delta \Phi, \delta P, \delta \zeta, \delta \tau\} = \int_{t_0}^{t_1} \delta \mathcal{L}\{\mathbf{u}, \Phi, P, \zeta, \tau : \delta \mathbf{u}, \delta \Phi, \delta P, \delta \zeta, \delta \tau\} dt.$$
(3.17)

From Eqs. (3.2-3.11), the first variation is evaluated as

224
$$\delta \mathcal{A} = -\int_{t_0}^{t_1} \iint_{\Omega_{\rm sh}} \left\{ \delta U \left(\rho_{\rm i} U_{tt} - \sigma_{11,x} - \sigma_{12,z} \right) \right\}$$

225
$$+ \delta W \left(\rho_{i} W_{tt} - \sigma_{21,x} - \sigma_{22,z} + \rho_{i} g \right) \bigg\} dx dz dt$$

226
$$+ \int_{t_0}^{t_1} \iint_{\Omega_{ca}} \left\{ \delta P \, \nabla^2 \Phi + \delta \Phi \, \nabla^2 \hat{P} \right\} \, dx \, dz \, dt$$

227
$$+ \int_{t_0}^{t_1} \iint_{\Omega_{op}} \left\{ \delta P \, \nabla^2 \Phi + \delta \Phi \, \nabla^2 \hat{P} \right\} \, \mathrm{d}x \, \mathrm{d}z \, \mathrm{d}t$$

228
$$-\int_{t_0}^{t_1}\int_0^\infty \left[\delta\zeta_{i-a}\,\rho_i\,g\,(W-\zeta_{i-a})\right]$$

$$+ \delta W \left(\sigma_{22} + \rho_{i} g \zeta_{i-a} + P_{at} \right) + \delta U \sigma_{12} \bigg]_{z=h-d} dx dt$$

230
$$+ \int_{t_0}^{t_1} \int_0^{h-d} \left[\delta W \,\sigma_{12} + \delta U \left(\sigma_{11} + P_{at} \right) \right]_{x=0} dz \, dt$$

231
$$+ \int_{t_0}^{t_1} \int_0^\infty \left[\delta \zeta_{W-i} \left\{ \rho_i g \left(W - \zeta_{W-i} \right) - \rho_W g \left(\Phi_z - \zeta_{W-i} \right) \right\} \right]$$

$$+ \delta W \left(\sigma_{22} + \rho_{i} g \zeta_{w-i} - S_{bt}\right) + \delta U \sigma_{12}$$

$$-\delta\Phi \hat{P}_{z} + \delta\Phi_{z} \left(P - \rho_{w} g \zeta_{w-i} + S_{bt}\right) - \delta S_{bt} \left(W - \Phi_{z}\right)\Big]_{z=-d} dx dt$$

234
$$+ \int_{t_0}^{t_1} \int_{-d}^{0} \left[\delta W \,\sigma_{12} + \delta U \left(\sigma_{11} - S_{\rm fr} \right) - \delta \Phi \, \hat{P}_x \right]$$

$$+ \,\delta \Phi_x \left(P + S_{\rm fr} \right) - \delta S_{\rm fr} \left(U - \Phi_x \right) \bigg|_{x=0} \,\mathrm{d}z \,\mathrm{d}t$$

$$+ \int_{t_0}^{t_1} \int_{-\infty}^{\infty} \left[\delta \Phi \hat{P}_z - \delta \Phi_z \left(P + S_{\text{bd}} \right) - \delta S_{\text{bd}} \Phi_z \right]_{z=-H} dx dt$$

220

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236
$$-\int_{t_0}^{t_1}\int_{-\infty}^0 \left[\delta\zeta_{\mathrm{W-a}}\,\rho_{\mathrm{W}}\,g\left(\Phi_z-\zeta_{\mathrm{W-a}}\right)+\delta\Phi\,\hat{P}_z\right]$$

237
$$-\delta\Phi_z \left(P - \rho_w g \zeta_{w-a} - P_{at}\right)\Big]_{z=0} dx dx$$

238
$$+ \int_{t_0}^{t_1} \int_{-H}^{-d} \left[\delta \Phi \hat{P}_x - \delta \Phi_x \left(P + S_{\rm fr} \right) \right]_{x=0^-}^{0^+} \mathrm{d}z \, \mathrm{d}t$$

239
240
$$-\int_{t_0}^{t_1} \int_{-H}^{-d} \left[\delta S_{\rm fr} \langle \Phi_x \rangle \right]_{x=0} dz dt.$$
(3.18)

Here, $\langle \bullet \rangle$ denotes the jump in the included quantity over x = 0, and the notations

242
$$\hat{P}(x, z, t) \equiv P + \rho_w (\Phi_{tt} + g z),$$
 $S_{fr}(z, t) \equiv [\tau_{11}]_{x=0},$ (3.19a,b)

$$\sum_{244}^{243} S_{bt}(x,t) \equiv [\tau_{22}]_{z=-d} \qquad \text{and} \quad S_{bd}(x,t) \equiv [\tau_{22}]_{z=-H}, \qquad (3.19c,d)$$

have been introduced for convenience, where the subscripts fr, bt and bd indicate stresses on the shelf front, shelf bottom and seabed, respectively. Vanishing of the first variations of the applied stresses from the atmosphere have been incorporated, as the stresses are known from (3.6) and (3.13). All variations are assumed to vanish in the far field $x \to \pm \infty$.

Enforcing $\delta \mathcal{A} = 0$ for arbitrary variations, $\delta \mathbf{u}$ and so on, \hat{P} must satisfy Laplace's equation

251
$$\nabla^2 \hat{P} = 0$$
 for $(x, z) \in \Omega_{op}$ and $(x, z) \in \Omega_{ca}$, (3.20)

252 (from domain integral terms proportional to $\delta \Phi$ in Eq. 3.18), with boundary conditions

253
$$\hat{P}_x = 0$$
 for $x = 0, -d < z < 0,$ $\hat{P}_z = 0$ for $-\infty < x < 0, z = 0,$ (3.21a,b)
254 $\hat{P}_z = 0$ for $0 < x < \infty, z = -d$ and $\hat{P}_z = 0$ for $-\infty < x < \infty, z = -H,$
255 (3.21c,d)

(from the terms proportional to $\delta \Phi$ in the respective boundary integrals). Eqs. (3.20–3.21) for \hat{P} are uncoupled from the other unknowns, and can be solved to give

258
$$\hat{P} = C_{\text{op}}(t)$$
 for $(x, z) \in \Omega_{\text{op}}$ and $\hat{P} = C_{\text{ca}}(t)$ for $(x, z) \in \Omega_{\text{ca}}$, (3.22a,b)

where C_{op} and C_{ca} are arbitrary functions.

Water pressures in Ω_{op} and Ω_{ca} can be deduced from Eqs. (3.22a,b), respectively. If we also set

 $C_{\rm op} = C_{\rm ca} \equiv P_{\rm at},\tag{3.23}$

(implicitly using the freedom of an arbitrary function of time in the potential Φ), the water pressure is given by as the sum of the hydrostatic pressure (introduced earlier) and a dynamic pressure, such that

266

262

$$P = P_{\text{at}} - \rho_{\text{w}} (\Phi_{tt} + g z) \quad \text{for} \quad (x, z) \in \Omega_{\text{op}} \cup \Omega_{\text{ca}}. \tag{3.24}$$

Therefore, Bernoulli's equation (3.24) appears as a natural condition of the variational principle, rather than it being imposed as an essential condition.

From the remaining conditions given by $\delta \mathcal{A} = 0$, it is possible to deduce the *field equations* of the full linear problem:

$\rho_{\rm i} U_{tt} = \sigma_{11,x} + \sigma_{12,z}$	for $(x, z) \in \Omega_{sh}$,	(3.25 <i>a</i>)
$\rho_{\rm i} W_{tt} = \sigma_{12,x} + \sigma_{22,z} - \rho_{\rm i} g$	for $(x, z) \in \Omega_{sh}$,	(3.25 <i>b</i>)
and $\nabla^2 \Phi = 0$	for $(x, z) \in \Omega_{ca} \cup \Omega_{op}$,	(3.25c)

where the continuities at the ocean-cavity interface $\langle \Phi \rangle = \langle \Phi_x \rangle = 0$ have been used. 272 Eqs. (3.25a,b) are the full equations of linear elasticity in the ice shelf, and Eq. (3.25c) is 273 Laplace's equation in the water, resulting from the standard assumptions of potential flow 274 theory. Further, $\delta \mathcal{A} = 0$ derives the *interfacial equations of the full linear problem*: 275

$$\begin{split} W &= \zeta_{i-a}, \quad \sigma_{12} = 0 \quad \text{and} \quad \sigma_{22} + \rho_i \, g \, \zeta_{i-a} = -P_{at} \quad \text{for} \quad 0 < x < \infty, \ z = h - d, \\ (3.26a,b,c) \end{split}$$

$$\sigma_{12} &= 0 \quad \text{and} \quad \sigma_{11} = -P_{at} \quad \text{for} \quad x = 0, \ 0 < z < h - d, \\ (3.26d,e) \end{split}$$

$$W &= \Phi_z = \zeta_{w-i}, \quad \sigma_{12} = 0, \quad \sigma_{22} + \rho_i \, g \, \zeta_{w-i} = S_{bt} \qquad \text{and} \quad P - \rho_w \, g \, \zeta_{w-i} = -S_{bt} \quad \text{for} \quad 0 < x < \infty, \ z = -d, \\ (3.26f,g,h,i) \end{aligned}$$

$$U &= \Phi_x, \quad \sigma_{12} = 0 \quad \text{and} \quad S_{fr} = -P = \sigma_{11} \quad \text{for} \quad x = 0, \ -d < z < 0, \\ (3.26j,g,h,i) \end{aligned}$$

$$\Phi_z = 0 \quad \text{and} \quad S_{bd} = -P \quad \text{for} \quad -\infty < x < \infty, \ z = -H, \\ (3.26m,n) \end{aligned}$$

276

Eq. (3.26) contains conditions at the interfaces between (a-e) the ice shelf and the atmosphere, 277 (f–l) the ice shelf and the water, (m–n) the water and the seabed, and (o–p) the water and the 278 atmosphere. Eqs. (3.26a,f,j,m,o) are kinematic conditions, i.e., matching of displacements at 279 common boundaries. Eqs. (3.26b,d,g,k) are continuities of shear stress (only non-zero in the 280 ice shelf), and Eqs. (3.26c,e,h,i,l,n,p) are continuities of normal stresses. Eq. (3.26n) is an 281 282 identity for the applied stress at the seabed, which may be evaluated once the other unknowns 283 have been calculated from the boundary value problem defined by the field equations (3.25)and the remaining interfacial conditions (3.26), plus radiation conditions. 284

4. Thin plate approximation 285

A thin plate (depth averaged) approximation for the ice shelf displacements, $\mathbf{u} = (U, W)$, is 286 derived using the ansatzes 287

288
$$U(x,z,t) \approx \overline{U}(x,t) - (z+d-h/2) \overline{W}_x(x,t) \text{ and } W(x,z,t) \approx \overline{W}(x,t), \quad (4.1a,b)$$

which include a simplified form of extensional motions, via \overline{U} , as well as flexural motion, 289 via \overline{W} . Eq. (4.1b) and the term proportional to \overline{W}_x in (4.1a) are the standard assumptions of 290 291 flexural waves in thin plates, i.e., points initially normal to the mid-plane (z = h/2 - d in

The components of the strain tensor (2.5b) reduce to

294
$$\varepsilon_{11} = \overline{U}_x - (z + d - h/2) \overline{W}_{xx}$$
 and $\varepsilon_{12} = \varepsilon_{21} = \varepsilon_{22} \equiv 0.$ (4.2a,b)

295 Thus, $\sigma_{12} = 0$, and assuming $\sigma_{22} = 0$ (i.e., plane stress), Eqs. (2.5b) and (4.2b) imply

$$\sigma_{33} = v \,\sigma_{11} \implies \sigma_{11} = M_{\rm ps} \,\varepsilon_{11}, \tag{4.3a,b}$$

where $M_{ps} = E / (1 - v^2)$ is the plane stress primary wave (P-wave) modulus. As noted by Fung (1965), the ansatz (4.1b) is technically inconsistent with the assumption $\sigma_{22} = 0$, since $\varepsilon_{22} = -v (1 + v) \sigma_{11} / E$, i.e., there should be an extension (contraction) in the *z*-direction whenever there is a contraction (extension) in the *x*-direction. This effect is neglected here, in order to follow the standard thin plate approximation.

Applying (4.1) in the ice shelf Lagrangian, \mathcal{L}_{sh} , the first variation of the associated action,

304
$$\mathcal{A}_{\rm sh} = \int_{t_0}^{t_1} \mathcal{L}_{\rm sh} \,\mathrm{d}t, \qquad (4.4)$$

305 becomes

306
$$\delta \mathcal{A}_{\rm sh} = -h \int_{t_0}^{t_1} \int_0^\infty \left\{ \delta \overline{U} \left(\rho_i \, \overline{U}_{t\,t} - M_{\rm ps} \, \overline{U}_{x\,x} \right) \right\} \, \mathrm{d}x \, \mathrm{d}t$$

307
$$-\int_{t_0}^{t_1} \int_0^\infty \left\{ \delta \overline{W} \left(\rho_i \, h \, \overline{W}_{t\,t} + \frac{h^3 \left\{ M_{\text{ps}} \, \overline{W}_{x\,x\,x\,x} - \rho_i \, \overline{W}_{x\,x\,t\,t} \right\}}{12} \right. \right\}$$

$$308 \qquad + g h \rho_i + S_{bt} + P_{at} + g \rho_i \left(\zeta_{i-a} - \zeta_{w-i}\right) \bigg\} dx dt$$

$$309 \qquad \qquad + \int_{t_0}^{t_1} \int_0^\infty \left\{ g \,\rho_i \,\delta\zeta_{\mathrm{W-i}} \left(\overline{W} - \zeta_{\mathrm{W-i}} \right) - g \,\rho_i \,\delta\zeta_{\mathrm{i-a}} \left(\overline{W} - \zeta_{\mathrm{i-a}} \right) \right\} \,\mathrm{d}x \,\mathrm{d}t$$

310
$$-\int_{t_0}^{t_1}\int_0^\infty \left\{\delta S_{\rm bt}\,\overline{W}\right\}\,\mathrm{d}x\,\mathrm{d}t$$

311
$$+ \int_{t_0}^{t_1} \left[\delta \overline{U} \left(h M_{\text{ps}} \overline{U}_x - \int_{-d}^{h-d} S_{\text{fr}} dz \right) \right]_{x=0} dt$$

312
$$+ \int_{t_0}^{t_1} \left[\delta \overline{W} \left(\frac{\rho_i h^3}{12} \overline{W}_{xtt} - \frac{h^3 M_{\text{ps}} \overline{W}_{xxx}}{12} \right) \right]_{x=0} dt$$

313
$$+ \int_{t_0}^{t_1} \left[\delta \overline{W}_x \left(\frac{h^3 M_{\text{ps}}}{12} \overline{W}_{xx} + \int_{-d}^{h-d} \left(d - \frac{h}{2} + z \right) S_{\text{fr}} dz \right) \right]_{x=0} dt$$

$$-\int_{t_0}^{t_1} \int_{-d}^{0} \left[\delta S_{\rm fr} \left(\overline{U} - \left(d - \frac{h}{2} + z \right) \overline{W}_x - \Phi_x \right) \right]_{x=0} dz dt.$$
(4.5)

316 Combining (4.5) with the relevant components of $\delta \mathcal{A}_{ca} = \int_{t_0}^{t_1} \delta \mathcal{L}_{ca} dt$, the vertical component

Rapids articles must not exceed this page length

317 of the shelf displacement is coupled to the cavity via the conditions

318
$$\overline{W} - \zeta_{i-a} = 0, \quad [\Phi_z]_{z=-d} - \overline{W} = 0, \quad P - \rho_w g \zeta_{w-i} + S_{bt} = 0, \quad (4.6a,b,c)$$

319
$$g \rho_i \left(h + \overline{W} - \zeta_{W-i} \right) - \left(P_0 + \rho_W g \left([\Phi_z]_{z=-d} - \zeta_{W-i} \right) \right) = 0, \tag{4.6d}$$

320 and
$$\rho_i h \overline{W}_{tt} + \frac{h^3 \{M_{ps} \overline{W}_{xxxx} - \rho_i \overline{W}_{xxtt}\}}{12}$$

$$\frac{321}{322} + g h \rho_i + S_{bt} + P_{at} + g \rho_i (\zeta_{i-a} - \zeta_{w-i}) = 0, \qquad (4.6e)$$

for x > 0. As $P_0 = \rho_i g h$, it follows from (4.6a,b,d) that

324
$$\overline{W} = \zeta_{w-i} = \zeta_{i-a} = [\Phi_z]_{z=-d} .$$
(4.7)

Substituting (4.7) into (4.6c,e), and using the Bernoulli pressure (3.24) and Archimedean draft, results in a thin plate equation for the ice shelf flexure, forced by the water motion (given below in Eq. 4.8a). In contrast, the thin plate equation for the extensional motion (from the first integral in Eq. 4.5) is not coupled to the cavity directly.

Therefore, the approximate $\delta \mathcal{A}_{sh}$ (in Eq. 4.5) combined with $\delta \mathcal{A}_{ca}$ derives the *thin plate* approximation field equations:

$$\rho_{w}\left([\Phi_{tt}]_{z=-d} + g\,\overline{W}\right) + \rho_{i}\,h\,\overline{W}_{t\,t} + \frac{h^{3}\left\{M_{ps}\overline{W}_{x\,x\,x\,x} - \rho_{i}\,\overline{W}_{x\,x\,t\,t}\right\}}{12} = 0, \quad (4.8a)$$

and $\rho_{i}\,\overline{U}_{t\,t} - M_{ps}\,\overline{U}_{x\,x} = 0, \quad (4.8b)$

331

for x > 0. Eq. (4.8a) is similar to the benchmark thin plate equation, i.e., a Kirchoff plate with fluid loading, but also contains rotational inertia, as with a Timoshenko-Mindlin plate (Fox & Squire 1991*a*; Balmforth & Craster 1999). Eq. (4.8b) is the standard field equation for extensional waves in an elastic plate that travel at the P-wave speed $\sqrt{M_{\rm ps} / \rho_{\rm i}}$, i.e., the extensional Lamb wave speed, consistent with Abrahams *et al.* (2023).

Coupling Eq. (4.5) with $\delta \mathcal{A}_{op} = \int_{t_0}^{t_1} \delta \mathcal{L}_{op} dt$, derives the *shelf front conditions for the thin-plate approximation:*

$$\frac{h^3 M_{\rm ps}}{12} \overline{W}_{x\,x} + \int_{-d}^{h-d} \left(d - \frac{h}{2} + z \right) S_{\rm fr} \, \mathrm{d}z = 0, \tag{4.9a}$$

$$M_{\rm ps}\,\overline{W}_{x\,x\,x} - \rho_i\,\overline{W}_{x\,t\,t} = 0,\tag{4.9b}$$

$$h M_{\rm ps} \overline{U}_x - \int_{-d}^{h-d} S_{\rm fr} \, \mathrm{d}z = 0, \qquad (4.9c)$$

and
$$\Phi_x - \left(\overline{U} - \left(d - \frac{h}{2} + z\right)\overline{W}_x\right) = 0$$
 for $-d < z < 0$, (4.9*d*)

339

for x = 0, where $S_{\text{fr}} = -[P]_{x=0}$ for $z \in (-d, 0)$ and $S_{\text{fr}} = -P_{\text{at}}$ for $z \in (0, h - d)$. Eqs. (4.9a–d) represent, respectively, continuity of bending moment, shear stress, normal

traction and horizontal displacement. As in the full linear problem, the potential Φ satisfies 342 Laplace's equation in the water domain, the impermeable seabed condition and the free 343 surface conditions, i.e., Eq. (3.25c) and Eqs. (3.26m,o,p).

5. Frequency domain 345

5.1. Governing equations and single-mode approximation

Assume all dynamic components are time-harmonic at some prescribed frequency, $\omega \in \mathbb{R}_+$, 347 so that the extensional and flexural components of the ice displacements are, respectively, 348

349
$$\overline{U}(x,t) = u(x) e^{-i\omega t}$$
 and $\overline{W}(x,t) = w(x) e^{-i\omega t}$, (5.1)

and the interfacial displacements are 350

351
$$\zeta_{\bullet}(x,t) = \eta_{\bullet}(x) e^{-i\omega t} \quad \text{for} \quad \bullet = w-a, w-i, i-a, \tag{5.2}$$

where $u, w, \eta_{\bullet} \in \mathbb{C}$ and it is implicitly assumed from here on that only the real parts are 352 353 retained for the time-dependent variables. For the water, prescribe Bernoulli pressure via

354
$$P(x, z, t) = P_{at} - \rho_w \{ \Phi_{tt} + g z \} \quad \text{for} \quad (x, z) \in \Omega_{op} \cup \Omega_{ca} \quad \Rightarrow \quad \hat{P} = P_{at}, \tag{5.3}$$

and constrain the vertical dependence of the potential, such that 355

356
$$\Phi(x, z, t) \approx \frac{g}{\omega^2} \varphi(x) \frac{\cosh\{k \ (z+H)\}}{\cosh(k \ H)} e^{-i \ \omega \ t} \quad \text{for} \quad (x, z) \in \Omega_{\text{op}}, \tag{5.4a}$$

$$\Phi(x,z,t) \approx \frac{g}{\omega^2} \psi(x) \frac{\cosh\{\kappa (z+H)\}}{\cosh\{\kappa (H-d)\}} e^{-i\omega t} \quad \text{for} \quad (x,z) \in \Omega_{ca},$$
(5.4b)

for wavenumbers $k, \kappa \in \mathbb{R}_+$ to be defined, i.e. a single-mode approximation (Porter & Porter 359 2004; Bennetts *et al.* 2007), noting that Eqs. (5.4a-b) create a jump in the potential over the 360 interface $z \in (-H, -d)$ for x = 0. The stresses at the shelf bottom and front are prescribed as 361 362

363
$$S_{bt}(x) = -[P]_{z=-d} + \rho_w g \zeta_{w-i} \qquad \text{for } x > 0 \qquad (5.5a)$$

364 and
$$S_{\rm fr}(z) = P_{\rm at} - \rho_{\rm w} \{ [\Phi_{tt}]_{x=0} + g z \}$$
 for $-H < z < 0.$ (5.5b)

Applying these constraints to $\delta \mathcal{A}$ in Eq. (3.18), using $\delta \mathcal{A}_{sh}$ in Eq. (4.5), gives 366

367
$$\delta \mathcal{A} = -h \int_{t_0}^{t_1} \int_0^\infty e^{-2i\omega t} \,\delta u \left\{ -\rho_i \,\omega^2 \,u - M_{\rm ps} \,u^{\prime\prime} \right\} \,\mathrm{d}x \,\mathrm{d}t$$

368
$$-\int_{t_0}^{t_1} \int_0^\infty e^{-2i\omega t} \,\delta w \left\{ -\rho_i \,h\,\omega^2 \,w + \frac{h^3 \left\{ M_{\rm ps} \,w^{\prime\prime\prime\prime} + \rho_i \,\omega^2 \,w^{\prime\prime} \right\}}{12} \right\}$$

369
$$+ g \rho_{w} (\eta_{w-i} - \psi) + g \rho_{i} (\eta_{i-a} - \eta_{w-i}) \bigg\} dx dt$$

370
$$+ \frac{\rho_{\rm w} g^2}{\omega^2} \int_{t_0}^{t_1} \int_0^\infty e^{-2\,\mathrm{i}\,\omega\,t} \,\delta\psi \left\{ \int_{-H}^{-d} (\psi'' + \kappa^2\,\psi) \,\frac{\cosh^2\{\kappa\,(z+H)\}}{\cosh^2\{\kappa\,(H-d)\}} \,\mathrm{d}z \right\}$$

$$+\left\{\frac{\omega^2}{g}w-\kappa\,\tanh\{\kappa\,(H-d)\}\psi\right\}\right\}\mathrm{d}x\,\mathrm{d}t$$

344

371
$$+ g \int_{t_0}^{t_1} \int_0^\infty e^{-2i\omega t} \,\delta\eta_{\text{w-i}} \left(\rho_i - \rho_w\right) \left(w - \eta_{\text{w-i}}\right) dx \,dt$$

372
$$-\rho_{i}g \int_{t_{0}}^{t_{1}} \int_{0}^{\infty} e^{-2i\omega t} \delta\eta_{i-a} (w - \eta_{i-a}) dx dt$$

373
$$+ \frac{\rho_{w} g^{2}}{\omega^{2}} \int_{t_{0}}^{t_{1}} \int_{-\infty}^{0} e^{-2i\omega t} \delta\varphi \left\{ \int_{-H}^{0} (\varphi'' + k^{2} \varphi) \frac{\cosh^{2}\{k (z+H)\}}{\cosh^{2}(k H)} dz \right\}$$

$$+ \tanh(k H) \left\{\varphi - \eta_{\text{w-a}}\right\} dx dt$$

375
$$-\frac{\rho_{\rm w} g^2}{\omega^2} \int_{t_0}^{t_1} \int_{-\infty}^0 e^{-2i\omega t} \,\delta\eta_{\rm w-a} \left(k \tanh(kH)\varphi - \frac{\omega^2}{g} \eta_{\rm w-i}\right) dx \,dz$$

$$\frac{376}{5.6}$$
 + $\delta C_{\text{op-ca}}$ + $\delta C_{\text{op-sh}}$, (5.6)

where $\delta C_{\text{op-ca}}$ and $\delta C_{\text{op-ca}}$ contain contributions on the interfaces between the open water and the shelf front and cavity, respectively.

Setting $\delta \mathcal{A} = 0$ for arbitrary variations (δu and so on) gives a set of governing equation for the unknown functions of the horizontal spatial coordinate in Eqs. (5.1–5.4), which includes depth-averaged equations in the open water and cavity. In the open water (x < 0)

383
$$a_{\rm op} \left(\varphi'' + k^2 \varphi\right) + \tanh(k H) \left\{\varphi - \eta_{\rm w-a}\right\} = 0$$
 (5.7a)

384 where
$$a_{\rm op} = \int_{-H}^{0} \frac{\cosh^2\{k \, (z+H)\}}{\cosh^2(k \, H)} \, \mathrm{d}z,$$
 (5.7b)

³⁸⁵
₃₈₆
$$k \tanh(k H) \varphi - \frac{\omega^2}{g} \eta_{w-a} = 0$$
 and $\varphi - \eta_{w-a} = 0$, (5.7c,d)

387 Eqs. (5.7c,d) imply

388
$$k \tanh(kH) = \frac{\omega^2}{g},$$
 (5.7e)

so that
$$k \in \mathbb{R}^+$$
 used in Eq. (5.4*a*) satisfies the standard *open water dispersion relation* (Fig. 2).
Therefore, $\delta \mathcal{A} = 0$ derives the *field equation of the open water single-mode approximation:*

 $\varphi'' + k^2 \varphi = 0 \text{ for } x < 0,$ (5.8)

392 which has the general solution

393
$$\varphi(x) = A^{(\text{op})} e^{i k x} + B^{(\text{op})} e^{-i k x}, \qquad (5.9)$$

394 for as yet unspecified constants $A^{(op)}$ and $B^{(op)}$.



Figure 2: Wavenumbers for the open water (k), flexural-gravity wave (κ) and extensional wave in the shelf (q) versus frequency for shelf thickness h = 200 m and water depth H = 800 m, along with the standard parameter values $\rho_i = 0.9 \rho_w$, E = 11 GPa, $\nu = 0.3$ and g = 9.81 m s⁻².

The depth-averaged equation in the cavity (x > 0) is

396

$$a_{\rm ca}\psi'' + \{\kappa^2 a_{\rm ca} - \kappa \tanh\{\kappa (H - d)\}\}\psi + \frac{\omega^2}{g}w = 0, \qquad (5.10a)$$

397
398 where
$$a_{ca} = \int_{-H}^{-d} \frac{\cosh^2{\{\kappa (z+H)\}}}{\cosh^2{\{\kappa (H-d)\}}} dz.$$
 (5.10b)

399 The remaining equations in the shelf-cavity involving the flexural shelf displacement are

$$400 - \rho_i h \omega^2 w + \frac{h^3 \{M_{\rm ps} w'''' + \rho_i \omega^2 w''\}}{12} + g \rho_{\rm w} (\eta_{\rm w-i} - \psi) + g \rho_i (\eta_{\rm i-a} - \eta_{\rm w-i}) = 0 \quad (5.11a)$$

401

402 $w = \eta_{i-a}$ and $(\rho_i - \rho_w)(w - \eta_{w-i}) = 0 \implies w = \eta_{w-i} = \eta_{i-a}.$ (5.11b,c,d) 403 Therefore, enforcing $\delta \mathcal{A} = 0$ derives the *coupled field equations of the single-mode and* 404 *thin-plate approximations*:

$$(1 - m\omega^2) w + F w'''' + J \omega^2 w'' - \psi = 0$$
 (5.12a)

$$a_{\rm ca}\psi'' + \{\kappa^2 a_{\rm ca} - \kappa \tanh\{\kappa (H - d)\}\}\psi + \frac{\omega^2}{g}w = 0, \qquad (5.12b)$$

and
$$G u'' + m \omega^2 u = 0$$
 (5.12c)

405

406 for x > 0, where

$$F \equiv \frac{M_{\rm ps} h^3}{12 \,\rho_{\rm w} g}, \quad G \equiv \frac{h \,M_{\rm ps}}{\rho_{\rm w} g}, \quad J \equiv \frac{\rho_{\rm i} \,h^3}{12 \,\rho_{\rm w} g} \quad \text{and} \quad m \equiv \frac{\rho_{\rm i} \,h}{\rho_{\rm w} g}. \tag{5.13}$$

Eqs. (5.12a,b) are identical to the single-mode approximation of Porter & Porter (2004) and Bennetts *et al.* (2007), except for the appearance of rotational inertia. Therefore, adapting Porter & Porter (2004) and Bennetts *et al.* (2007) to include rotational inertia, the general 411 solutions are

412
$$\psi(x) = A^{(ca)} e^{i\kappa x} + B^{(ca)} e^{-i\kappa x} + \sum_{n=1,2} \left\{ A^{(ca)}_{-n} e^{i\kappa_{-n}x} + B^{(ca)}_{-n} e^{-i\kappa_{n}x} \right\}$$
(5.14*a*)

413 and
$$w(x) = A^{(\text{fl})} e^{i\kappa x} + B^{(\text{fl})} e^{-i\kappa x} + \sum_{n=1,2} \left\{ A_{-n}^{(\text{fl})} e^{i\kappa_{-n}x} + B_{-n}^{(\text{fl})} e^{-i\kappa_{-n}x} \right\},$$
 (5.14b)
414

415 for as yet unspecified constants $A^{(ca)}$, $B^{(ca)}$, $A^{(fl)}$ and $B^{(fl)}$, such that

416
$$A^{(ca)} = \frac{\omega^2}{g \kappa \tanh\{\kappa (H-d)\}} A^{(fl)}$$
(5.15*a*)

417 and
$$A_{-n}^{(ca)} = a_{ca}^{-1} \{F(\kappa^2 + \kappa_{-n}^2) - J\omega^2\} \kappa \tanh\{\kappa (H-d)\} A_{-n}^{(fl)} \quad (n = 1, 2), \quad (5.15b)$$

and similarly for the constants related to the left-going waves. The wavenumber κ is a root of the *flexural-gravity wave dispersion equation*

421
$$\{F\kappa^4 - J\omega^2\kappa^2 + 1 - m\omega^2\}\kappa \tanh\{\kappa (H - d)\} = \frac{\omega^2}{g}.$$
 (5.16)

For low frequencies, the flexural-gravity wavenumber, κ , is similar to the open-water wavenumber, k, as restoring due to flexure (and rotational inertia) are negligible, but is slightly larger due to the reduced water depth, i.e., H - d < H (Fig. 2). For high frequencies, flexural restoring dominates and the flexural-gravity wavenumber becomes much smaller than the open water wavenumber. The wavenumbers $\kappa_{-n} \in \mathbb{R} + i \mathbb{R}^+$ (n = 1, 2) are roots of the quartic equation

428
$$a_{ca} \left(F \kappa_{-n}^4 - J \omega^2 \kappa_{-n}^2 + 1 - m \omega^2 \right) + \left\{ F \left(\kappa^2 + \kappa_{-n}^2 \right) - J \omega^2 \right\} \kappa \tanh\{\kappa \left(H - d \right)\} = 0, (5.17)$$

which typically satisfy $\kappa_{-2} = -\kappa_1^*$, where * denotes the complex conjugate (Williams 2006; Bennetts 2007).

431 Eq. (5.12c) for the extensional component of the shelf motions has the general solution

432
$$u(x) = A^{(ex)} e^{i q x} + B^{(ex)} e^{-i q x}, \qquad (5.18)$$

433 for as yet unspecified constants $A^{(ex)}$ and $B^{(ex)}$. The extensional wavenumber, q, is

434
$$q = \omega \sqrt{\frac{m}{G}},$$
 (5.19)

which is typically much smaller than the flexural-gravity wavenumber (and the open water wavenumber; Fig. 2).

437 The contribution to $\delta \mathcal{A}$ on the interface between the open ocean and the cavity is

438
$$\delta C_{\text{op-ca}} = -\rho_{\text{w}} g \int_{t_0}^{t_1} e^{-2i\omega t} [\delta\varphi]_{x=0} \Biggl\{ \int_{-H}^0 [\varphi']_{x=0} \frac{\cosh^2\{k (z+H)\}}{\cosh^2(k H)} dz \Biggr\}$$

439
$$-\int_{-H}^{-d} [\psi']_{x=0} \frac{\cosh\{k \ (z+H) \ \cosh\{\kappa \ (z+H)\}}{\cosh(k \ H) \cosh\{\kappa \ (H-d)\}} \, \mathrm{d}z$$

440
$$-\int_{-d}^{0} \frac{\omega^2}{g} \frac{\cosh\{k \left(z+H\right)\}}{\cosh\{\kappa \left(H-d\right)\}} \left\{u - \left(d - \frac{h}{2} + z\right)w'\right\} dz \right\} dt$$

441
$$+ \int_{t_0}^{t_1} e^{-2i\omega t} \left[\delta\psi'\right]_{x=0} \left\{ \int_{-H}^0 [\varphi]_{x=0} \frac{\cosh^2\{\kappa (z+H)\}}{\cosh^2\{\kappa (H-d)\}} dz \right\}$$

442
443
$$-\int_{-H}^{-d} [\psi']_{x=0} \frac{\cosh\{k \ (z+H) \ \cosh\{\kappa \ (z+H)\}}{\cosh\{k \ (H-d)\}} \ dz \bigg\} dt. \quad (5.20)$$

444 Setting $\delta C_{\text{op-ca}} = 0$ leads to the *interfacial "jump" conditions for single-mode approximation:*

445
$$a_{\text{op-ca}} \varphi = a_{\text{ca}} \psi \quad \text{and} \quad a_{\text{op}} \varphi' = a_{\text{op-ca}} \psi' + \frac{\omega^2}{g} \left\{ v_0 \, u - v_1 \, w' \right\}$$
(5.21a,b)

446 for x = 0, where

448

447
$$a_{\rm op-ca} = \int_{-H}^{-d} \frac{\cosh\{k \ (z+H)\} \ \cosh\{\kappa \ (z+H)\}}{\cosh\{k \ (H-d)\}} \ dz, \tag{5.22}$$

$$v_0 = \int_{-d}^0 \frac{\cosh\{k \, (z+H)\}}{\cosh(k \, H)} \, \mathrm{d}z \tag{5.23}$$

449
450 and
$$v_1 = \int_{-d}^0 \frac{\cosh\{k \ (z+H)\}}{\cosh(k \ H)} \left(d - \frac{h}{2} + z\right) dz.$$
 (5.24)

451 Eq. (5.21a) is a weak form of continuity of pressure between the open ocean and subshelf water cavity. Eq. (5.21b) is a weak form of continuity of horizontal water velocity 452 between the open ocean and combined water and shelf front. The jump conditions are 453 identical to the jump conditions derived by Porter & Porter (2004) and Bennetts et al. (2007) 454 (restricted to piecewise constant geometry), except that (i) the integration of the coefficient 455 v_{op} extends to the free surface (z = 0) rather than the ice underside (z = -d), and (ii) the ice 456 displacements appear in Eq. (5.21b). For low frequencies, the (normalised) coefficient of the 457 cavity water velocity in Eq. (5.21b) is much greater than the (normalised) coefficients of the 458 ice displacement/velocity (Fig. 3a), indicating the jump condition is dominated by the depth 459 averaged water velocities. The coefficients of the ice displacement/velocity increase with 460 frequency, whereas the coefficient of water velocity decreases, such that the former become 461 462 comparable and then much greater than the latter, which indicates the jump condition provides strong coupling between the open ocean and shelf. 463



Figure 3: Normalised coefficients of the (a) jump condition Eq. (5.21b) and (b) shelf edge conditions Eqs. (5.26a,b), which couple the open water to the shelf, versus frequency, for ice thickness h = 200 m (thin dashed curves) and h = 400 m (thick solid) and water depth H = 800 m. Coefficients are normalised with respect to the coefficients of the relevant leading term. Appropriate wavenumbers replace the derivatives and (5.15a) is used to relate the amplitude of the flexural wave with the displacement potential.

The contribution to $\delta \mathcal{A}$ on the interface between the open ocean and the ice shelf is 464

465
$$\delta C_{\text{op-sh}} = \int_{t_0}^{t_1} [\delta u]_{x=0} \left\{ e^{-2i\omega t} \left(h M_{\text{ps}} [u']_{x=0} + \rho_{\text{w}} g [\varphi]_{x=0} \int_{-d}^{0} \frac{\cosh\{k (z+H)\}}{\cosh(k H)} dz \right) \right\}$$

$$+ e^{-i\omega t} \left(\int_{-d}^{0} P_{at} - \rho_{w} g z dz \int_{-d}^{0} P_{at} dz \right) \right\} dt$$

467
$$+ \int_{t_0}^{t_1} e^{-2i\omega t} [\delta w]_{x=0} \left(\frac{-\rho_i h^3 \omega^2}{12} [w']_{x=0} - \frac{h^3 M_{\text{ps}}}{12} [w''']_{x=0} \right) dt$$

468
$$+ \int_{t_0}^{t_1} [\delta w']_{x=0} \left\{ e^{-2i\omega t} \left(\frac{h^3 M_{\text{ps}}}{12} [w'']_{x=0} \right) \right\} \right\}$$

469
$$-\rho_{\rm w} g \left[\varphi\right]_{x=0} \int_{-d}^{0} \left(d - \frac{h}{2} + z\right) \frac{\cosh\{k \left(z + H\right)\}}{\cosh(k H)} dz$$

470
$$+ e^{-i\omega t} \left(\int_{-d}^{0} \left(d - \frac{h}{2} + z \right) \left(P_{at} - \rho_w g z \right) dz \int_{-d}^{0} \left(d - \frac{h}{2} + z \right) P_{at} dz \right) \right\} dt.$$
(5.25)

472 Setting $\delta C_{\text{op-sh}} = 0$ leads to the (dynamic, $\omega \neq 0$) shelf front conditions:

 $G u' + v_0 \varphi = 0$, $F w'' + v_1 \varphi = 0$ and $F w''' + J \omega^2 w' = 0$ (5.26a,b,c)

473

482

474 for x = 0. (The static conditions are given in Appendix B.) Eqs. (5.26a,b) couple the ice and open water displacements. The ratios of the coefficients in the coupling conditions (Fig. 3b) 475 indicate (i) strong coupling in Eq. (5.26a) at low frequencies $(-1 < \log_{10} |v_0| / (qG)| < 0)$ 476 for both thicknesses when $\log_{10}(\omega/(2\pi)) < -1.5$) degenerating to uncoupled zero normal 477 traction at high frequencies $(\log_{10} |v_0 / (q G)| < -2$ for $\log_{10}(\omega / (2\pi)) > -0.6)$, and 478 (ii) Eq. (5.26b) is approximately the bending moment component of the standard (uncoupled) 479 free edge conditions over the frequency range considered $(\log_{10} | \omega^2 v_1 / (g F \kappa^3 \tanh{\kappa} (H - \kappa)))$ 480 d)})| < -1, except for the thinner shelf over a short interval at low frequencies). 481

5.2. Scattering matrix

The jump conditions (5.21) and shelf front conditions (5.26) are applied to the general solutions (5.9) and (5.14) to derive a system of relations between the amplitudes of the waves that propagate/decay towards and away from x = 0, $A^{(\bullet)}$ and $B^{(\bullet)}$, respectively. Restricting to propagating waves only, and using Eq. (5.15) to eliminate $A^{(ca)}$ and $B^{(ca)}$, derives the scattering matrix, S, which relates the outgoing amplitudes to the incoming amplitudes, such that

$$489 \qquad \begin{pmatrix} B^{(op)} \\ B^{(f)} \\ B^{(ex)} \end{pmatrix} = S \begin{pmatrix} A^{(op)} \\ A^{(f)} \\ A^{(ex)} \end{pmatrix} \quad \text{where} \quad S = \begin{pmatrix} \mathcal{R}^{(op \to op)} & \mathcal{T}^{(fl \to op)} & \mathcal{T}^{(ex \to op)} \\ \mathcal{T}^{(op \to fl)} & \mathcal{R}^{(fl \to fl)} & \mathcal{R}^{(ex \to fl)} \\ \mathcal{T}^{(op \to ex)} & \mathcal{R}^{(fl \to ex)} & \mathcal{R}^{(ex \to ex)} \end{pmatrix}, \quad (5.26a,b)$$

in which the \mathcal{R}^{\bullet} and \mathcal{T}^{\bullet} are, respectively, reflection and transmission coefficients to be found from the solution of the problem in § 5.1. In general, $\mathcal{T}^{(op \to ex)} \neq \mathcal{T}^{(ex \to op)}$, etc, as $\mathcal{T}^{(op \to ex)}$ is the coefficient of the extensional wave in the ice shelf forced by a unit-amplitude incident wave from the open ocean, whereas $\mathcal{T}^{(ex \to op)}$ denotes the amplitude of a wave transmitted into the open ocean by an incident extensional wave from the ice shelf. The latter is typically not a physical problem considered in wave–shelf interaction studies. Using standard methods (Porter & Porter 2004), it can be deduced that

 $\mathcal{S}\,\mathcal{S}^* = \mathcal{I}\,,\tag{5.27}$

where * denotes the conjugate matrix and \mathcal{I} is the 3 × 3 identity matrix, from which energy balances can be derived (see below).

500 **6. Results**

Consider the problem in which motions are excited by an ambient incident wave from the ocean $(A^{\text{(fl)}} = A^{(\text{ex})} \equiv 0)$ at a prescribed period $T = 2\pi / \omega$. Without loss of generality, a unit incident wave amplitude is set $(A^{(\text{op})} = 1 \text{ m})$. The primary quantity of interest is the spatial component of the (non-zero) strain component

505
$$\hat{\varepsilon}_{11}(x, z:T) = u' - (z + d - h/2) w'', \tag{6.1}$$

which is such that $\varepsilon_{11}(x, z, t) = \hat{\varepsilon}_{11}(x, z) e^{-i\omega t}$. Examples of the strain field due to incident waves (Fig. 4) indicate that the extensional and flexural motions both contribute to the strain for relatively short periods (in the swell regime), as it has nonlinear structure in both spatial dimensions, whereas only the flexural motion contributes for longer periods (infragavity wave regime and above), indicated by the vertical symmetry about the unstrained mid-plane



Figure 4: Wave-induced strain fields up to 5 km from the shelf front, for ice thickness h = 200 m, water depth H = 800 m, and wave period (a) T = 15 s and (b) T = 50 s.

(z = h/2 - d). The shelf front experiences strains comparable to the shelf interior for the shorter period and near-zero strain for the longer period, where the latter is ensured by the exponentially decaying components of the flexural motion (with wavenumbers κ_{-n} in Eq. 5.14b).

Example wave-induced strain profiles at the lower ice shelf surface (Fig. 5) show the 515 influence of the additional terms in the thin-plate approximation. Results from the benchmark 516 thin plate model (without water-ice coupling at the shelf front and extensional waves) are 517 shown alongside results from an intermediate version of the model derived in § 5 that includes 518 water-ice coupling at the shelf front but no extensional waves, and the full model that includes 519 extensional waves. The differences between the intermediate model (with water-ice coupling) 520 and the full model (with extensional waves) highlight the influence of the extensional waves 521 522 on the shelf strains. The differences between the benchmark model (in which hydrodynamic loads are imposed only at the lower shelf surface) and the two new models highlight the 523 influence of hydrodynamic forcing at the shelf front on the shelf strains. In particular, the 524 differences between the benchmark and intermediate models isolate the effects of water-ice 525 coupling at the shelf front from the coupling at the lower surface on flexural waves. The 526 strains are scaled by the shelf thickness, such that strains for different thickness values are 527 528 of the same order of magnitude for the different wave periods. In all four cases (Fig. 5a–d), the benchmark model predicts the strain modulus increases from zero at the shelf front to a 529 maximum value after several kilometres, followed by a plateau at a slightly smaller value. 530

For the shorter wave period (Fig. 5a,b), the addition of water-ice coupling at the shelf 531 front (through Eqs. 5.21b and 5.26b) causes a large relative increase in the strain, by factors 532 533 of \approx 3 for the thinner shelf and \approx 75 for the thicker shelf at the plateaus (approximately x > 3 km). The strain at the shelf front is non-zero and, for the thicker shelf (Fig. 5b), the 534 greatest strain occurs at the shelf front, such that the wave-ice coupling causes a qualitative 535 change in the strain profile. The effect of wave-ice coupling on the strain profiles is almost 536 imperceptible for the longer wave period (Fig. 5c,d), although the strains are one to two 537 orders of magnitude larger than for the shorter period ($h | \hat{\epsilon}_{11} |$ is up to order 10^{-3} for T = 50 s versus order 10^{-5} – 10^{-4} for T = 15 s). Moreover, the change in scale masks the similarity 538 539



Figure 5: Comparison of scaled wave-induced strain profiles predicted by three thin plate models: (i) the benchmark model without water–ice coupling at the shelf front and extensional waves (Porter & Porter 2004; Bennetts *et al.* 2007); (ii) an intermediate model in which water–ice coupling occurs at the shelf front through the velocity jump condition (Eq. 5.21b) and the bending moment condition (Eq. 5.26b); and (iii) the full model proposed in § 5 including extensional wave motion and water–ice coupling at the shelf front, for shelf thickness (a,c) h = 200 m and (b,d) h = 400 m, and bed depth H = 800 m, in response to incident waves with period (a,b) T = 15 s and (c,d) T = 50 s.

in the shelf front strain values for the respective thicknesses, as anticipated by the couplingcoefficient in the bending moment condition (Fig. 3b; yellow curves).

The addition of extensional waves changes the qualitative behaviour of the strain profiles for the shorter wave period (Fig. 5a,b). Notably, the strain does not reach a constant value away from the shelf front, due to interference in the underlying wave field between the flexural wave (with wavenumber κ) and the extensional wave (*q*), both of which persist to the far-field, $x \to \infty$. The extensional waves have a far smaller effect on the strain profiles for the longer wave period (Fig. 5c,d), although their influence for the thicker shelf (Fig. 5d) is greater than that of the wave–ice coupling on the flexural waves.

The proportion of incident wave energy transmitted into the flexural and extensional waves is used to assess their relative influence on the ice shelf motion versus wave period. The distribution of incident wave energy is derived from Eq. (5.27), which gives the energy balance

$$\mathcal{R} + \mathcal{T}^{(\mathrm{fl})} + \mathcal{T}^{(\mathrm{ex})} = 1, \tag{6.2}$$

553

555
$$\mathcal{R} = |\mathcal{R}^{(\text{op}\to\text{op})}|^2, \ \mathcal{T}^{(\text{fl})} = |\mathcal{T}^{(\text{fx}\to\text{op})}| \ |\mathcal{T}^{(\text{op}\to\text{fl})}| \text{ and } \mathcal{T}^{(\text{ex})} = |\mathcal{T}^{(\text{ex}\to\text{op})}| \ |\mathcal{T}^{(\text{op}\to\text{ex})}|, \quad (6.3)$$

are the proportions of the incident energy in the reflected wave (\mathcal{R}), and the flexural ($\mathcal{T}^{(ff)}$) and extensional ($\mathcal{T}^{(ex)}$) waves transmitted into the shelf–cavity region.

For periods in the majority of the swell regime (here defined as wave periods from 10-30 s), the transmitted extensional waves carry more energy than the flexural waves (Fig. 6).



Figure 6: Transmitted energy proportions for flexural and extensional waves (blue and red curves, respectively) versus wave period, for shelf thickness (a) h = 200 m and (b) h = 400 m, and bed depth H = 800 m.

560 The difference is approximately an order of magnitude for the shortest periods considered, and is greatest for the thinner shelf (Fig. 6a). The proportion of energy in the flexural waves 561 increases steeply as wave period transitions from the swell to infragravity regimes, whereas 562 the proportion of energy in the extensional waves slightly decreases. This causes the flexural 563 wave energy to exceed the extensional wave energy in the infragravity wave regime, with the 564 difference approximately two orders of magnitude at the longest wave periods considered 565 and greater for the thinner shelf. The wave period at which the energies of the flexural and 566 extensional waves are equal is longer for the thicker shelf than the thinner shelf (≈ 30 s vs. 567 ≈ 23 s). 568

For the cases tested with the full approximation outlined in §5, the maximum shelf strains 569 due to incident waves are attained at either the upper or lower shelf surface, which is similar 570 to the benchmark model, where the maximum strains are attained at both upper and lower 571 surfaces due to symmetry about the mid-plane. In the swell regime, the maximum strains 572 predicted by the full model far exceed those of the benchmark model (Fig. 7a,b). The 573 maximum strains at the upper surfaces slightly exceed those at the lower surface for the 574 smallest wave periods considered. For longer periods, the maximum strains at the upper and 575 lower surfaces are almost identical, and they tends towards the maximum strain predicted by 576 the benchmark model, as the wave period increases, such that they are indistinguishable in 577 578 the infragravity regime.

For the shortest wave period considered, the strain maxima at the upper surface occur only hundreds of metres from the shelf front, and move closer towards the shelf front as wave period decreases (Fig. 7c,d). In contrast, the maxima predicted by the benchmark model occur more than a kilometre away from the shelf front, and the maxima at the lower surface predicted by the full approximation occur even farther away. For shorter periods and the thinner shelf, the strain maxima move between distinct regions of large strain at the upper



Figure 7: (a,b) Maximum flexural strains due to incident waves at upper (z = h - d) and lower (z = -d) shelf surfaces, and (c,d) corresponding locations, for shelf thickness (a,c) h = 200 m and (b,d) h = 400 m, and bed depth H = 800 m, with results of benchmark model shown for reference.

and lower surfaces (yellow patches in Fig. 4a), which causes the jumps in the locations of maximum strain (Fig. 7c).

587 7. Conclusions and discussion

The governing equations for the canonical problem of incident waves from the open ocean 588 forcing motions of a floating ice shelf, in which the ice shelf is modelled by the full equations 589 590 of elasticity and has an Archimedean draught, have been derived from a variational principle. The variational principle was used to derive a thin-plate approximation for the ice shelf. 591 Previous derivations of the governing equations for ice shelves (or other floating bodies) as 592 thin floating elastic plates, including those based on variational principles (Porter & Porter 593 2004; Bennetts et al. 2007), have assumed the thin plate approximation from the outset (i.e., 594 depth averaging in the ice), thus resulting in the ice shelf satisfying free edge conditions at 595 596 the shelf front. In contrast, the variational principle presented in this study derives shelf front conditions in which the water and ice are coupled. The water-ice coupling allows extensional 597 waves to be excited in the shelf, further extending previous thin plate approximations. The 598 thin-plate approximation was combined with a single-mode approximation in the water. 599 Results have shown that the water-ice coupling at the submerged portion of the shelf front 600 and the extensional waves significantly increase wave-induced shelf strains for wave periods 601 in the swell regime. In contrast, they have a negligible effect for periods in the infragravity 602 wave regime. 603

Variational principles are often used to derive approximations for water wave problems, dating back to Luke (1967), and with the so-called (modified) mild-slope equations of Miles (1991), Chamberlain & Porter (1995) and others of particular relevance to the present study. The variational principle presented in § 3 can be viewed as an extension of the variational

principle of Porter & Porter (2004) to incorporate the full equations of elasticity for the 608 floating ice. However, there are notable differences in the approach used here, which is 609 arguably more closely aligned to the 'unified theory' of Porter (2020) for open water waves. 610 In particular, our use of a displacement potential in the water, for consistency with the 611 unknown displacements in the ice, is a major departure from Porter & Porter (2004) and 612 others before. Further, we include interfacial stresses in the variational principle, so that all 613 614 matching conditions arise as natural conditions of the variational principle, and essential conditions do not have to be imposed. 615

There is evidence from studies on cognate problems (without water-ice coupling at the ice 616 edge and extensional waves) that the single-mode approximation is accurate (Bennetts et al. 617 2007, 2009; Bennetts & Meylan 2021). In particular, Liang et al. (2023) give evidence the 618 619 single-mode approximation is accurate for ice shelf strains across a range of relevant wave periods and for realistic geometries. However, in general, the single-mode approximation 620 becomes less accurate as frequency increases and the impedance mismatch between the open 621 water and the cavity water becomes more pronounced (Fig. 2). Following Bennetts et al. 622 (2007), the single-mode approximations (Eq. 5.4) can be extended to include an arbitrary, 623 finite number of vertical modes that support evanescent waves, such that continuities between 624 the open ocean and sub-shelf water cavity are satisfied to a desired accuracy. 625

The primary motivation for present study was to derive a consistent thin plate approx-626 imation, in which the water and ice are coupled at the shelf front. Regimes have been 627 found in which the water-ice coupling has a major impact on ice shelf strains. However, 628 studies are still needed to test the validity of the thin plate assumptions (Eq. 4.1 or similar), 629 particularly for thick shelves and incident swell. The studies could be based on numerical 630 solutions, for which the software presented by Kalyanaraman et al. (2021) is available if 631 the present model is modified to a finite length shelf and the gravitational acceleration in 632 the shelf is removed. Alternatively, similarly to the approach proposed above to extend the 633 single-mode approximation in the water, the thin plate ansatzes (Eq. 4.1) could be extended 634 with additional terms to improve accuracy. For instance, higher order terms in the ansatz for 635 the vertical displacement would remove an inconsistency between the low-order ansatz used 636 in this study and the plane stress assumption (Fung 1965). Therefore, the method outlined 637 to derive the thin plate approximation provides a framework to obtain the full solution 638 (Eqs. 3.25-3.26). 639

640 The approximation derived in this study (\S 5) predicts extensional wave displacements that are greater than flexural wave displacements for low frequencies (long periods), and that the 641 amplitude ratio becomes unbounded as frequency tends to zero, such that $\mathcal{T}^{(op \to ex)} \to \infty$ 642 as $\omega \to 0$ (not shown), which is consistent with the findings of Abrahams *et al.* (2023). 643 This property is a consequence of the elliptical trajectories of water particles created by the 644 incident waves having aspect ratios that increasingly skew towards the horizontal axis as 645 wavelengths increase. However, our results show that extensional waves have a negligible 646 impact on shelf strains for long periods (e.g., Fig. 5). Further, flexural waves hold greater 647 energy than extensional waves for long periods, i.e., $\mathcal{T}^{(fl)} \gg \mathcal{T}^{(ex)}$ for $T \gg 1$ (Fig. 6), where 648 the small limiting values of $\mathcal{T}^{(fl)}$ are due to decreases in $\mathcal{T}^{(ex\to op)}$ compensating for increases 649 in $\mathcal{T}^{(\text{op}\to\text{ex})}$. Intuitively, as incident waves get longer, the impact of the ice cover decreases, 650 resulting in $\kappa \approx k$ (Fig. 2) and most of the incident wave transmitting into a flexural-gravity 651 wave in the shelf-cavity interval ($\mathcal{T}^{(\text{op}\rightarrow\text{ex})} \approx 1$). 652

The strain magnitudes presented in § 6 are one to two orders of magnitude smaller for wave periods in the swell regime than in the infragravity wave regime. However, swell amplitudes are typically much greater than infragravity wave amplitudes, such that the benchmark model predicts they create strains of comparable magnitude (Bennetts *et al.* 2022). In particular, flexural-gravity waves at periods in the swell regime are amplified by crevasses in ice shelves (Bennetts *et al.* 2022), and, thus, our findings highlight a potential, additional role of water–
ice coupling and extensional waves in these amplifications. Further, the thin plate model
presented could be extended to study whether periodic thickness variations in the ice shelf
blocks incident ocean wave energy from propagation through the shelf (Freed-Brown *et al.*2012; Nekrasov & MacAyeal 2023).

The dynamic problem ($\omega \neq 0$) was considered in this study, so that the derived 663 approximation could be compared with the benchmark thin plate approximation, in order to 664 identify the influence of water-ice coupling at the shelf front and extensional waves. The 665 static problem ($\omega = 0$) can also be derived from the variational principle (Appendix B). Static 666 extensions are forced by traction at the shelf front (B 2c), due to atmospheric pressure and 667 static water pressure (B 3). The shelf front condition indicates the extensional contribution 668 to the non-zero strain is $u'(0) = -p_0 / G$, which is order 10^{-5} for h = 200 m and 400 m. It 669 is comparable to the dynamic strains induced by infragravity waves (Fig. 5c,d) and one to 670 two orders of magnitude greater than the dynamic strains induced by swell (Fig. 5a,b), 671 although these are only for one metre amplitude incident waves and mean daily swell 672 673 amplitudes reaching ice shelves can be up to four-five times larger (Teder et al. 2022). However, bounded static extensions require a finite shelf (B6). In contrast, the semi-674 infinite shelf supports bounded static flexure (B 5) forced by bending at the shelf front 675 (B 2a) due to static water pressure (B 3). The flexural contribution to the non-zero strain is 676 $(z+d-h/2)w''(0) = (z+d-h/2)p_1/F$, which have maximum values at z = -d on the 677 order of 10^{-5} for h = 200 m and 10^{-4} for h = 400 m. Therefore, the static problem indicates 678 the static strains close to the ice edge can be comparable or larger than the dynamics strains 679 caused by wave motion. This may motivate future studies to consider interactions between 680 the static and dynamic problems, i.e., pre-stress. Pre-stress on the relatively short time scales 681 of ocean waves could also result from long time scale viscous creep (e.g., Weertman 1957). 682

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686 Appendix A. Finite deformation of an infinitely long ice shelf

Consider an infinitely long ice shelf of constant density ρ_i in the absence of gravity. Let gravity be increased from zero, such that is compresses the ice shelf onto an incompressible water base. This induces a finite initial stress (typically called a "pre-stress") to the ice, which can modify the properties of waves in the ice.

In Eulerian coordinates (relative to the ice shelf after being compressed), the strain tensor is $\varepsilon^{\text{finite}}$, with components

$$\varepsilon_{ij}^{\text{finite}} = \varepsilon_{ij} - \frac{1}{2} \sum_{k=1}^{3} U_{k,x_i} \, u_{k,x_j} \tag{A1}$$

(Spencer 2004), where ε_{ij} is the linearised strain tensor from (2.4). In order to consider infinitesimal waves in the *x*-*z* plane, the displacement is split, such that

697
$$U_1(x, z, t) = \hat{U}(x, z, t), \quad U_2(x, z, t) = W^{hs}(z) + \hat{W}(x, z, t) \text{ and } U_3 = 0, \quad (A 2a,b,c)$$

where the superscript hs indicates hydrostatic displacements and hats indicate dynamic displacements due to waves. Substituting (A 2a,b,c) into (A 1), and ignoring second-order

693

terms involving \hat{u}_i , the strain is split into

$$\boldsymbol{\varepsilon}^{\text{finite}}(x, z, t) = \boldsymbol{\varepsilon}^{\text{hs}}(z) + \hat{\boldsymbol{\varepsilon}}(x, z, t), \tag{A3}$$

702 where

701

703
$$\varepsilon_{22}^{\text{hs}} = W_z^{\text{static}} \left(1 - \frac{1}{2} W_z^{\text{static}} \right) \text{ and } \hat{\varepsilon} = \frac{1}{2} \begin{pmatrix} 2\hat{U}_x & 0 & \hat{U}_z + \gamma \, \hat{W}_x \\ 0 & 0 & 0 \\ \hat{U}_z + \gamma \, \hat{W}_x & 0 & 2 \gamma \, \hat{W}_z \end{pmatrix}, \quad (A \, 4a, b)$$

704 in which

705 $\gamma(z) = 1 - W_z^{\text{hs}}$ and $\varepsilon_{ij}^{\text{hs}} = 0$ if $i \neq 2$ or $j \neq 2$ $(i, j \in \{1, 2, 3\})$. (A 5a,b)

The factor γ induces coupling between the static and wave problems. The density after compression is $\rho_i \gamma(z)$, i.e., it is no longer constant. However, if $W_z^{hs} \ll 1$, $\gamma \approx 1$, hence the coupling between the static and wave problems is removed and the ice has constant density. While the finite-deformation problem in this case is tractable, it is simpler and more instructive to solve the linear problem and check the size of W_z^{hs} *a posteriori*. From (3.25) and (3.26), the static problem can be written

$$\sigma_{22,z}^{\rm hs} = \rho_{\rm i} g, \qquad (A \, 6a)$$

$$\sigma_{22}^{\rm hs} = M W_z^{\rm hs}, \tag{A6b}$$

714
$$\sigma_{22}^{\rm hs}(h-d) + \rho_{\rm i} g W^{\rm hs}(h-d) = -P_{\rm at}, \qquad (A \, 6c)$$

$$715 W^{hs}(-d) = 0, (A 6d)$$

717 where

718

$$M = \frac{E(1-\nu)}{(1+\nu)(1-2\nu)}$$
(A7)

is the P-wave modulus, which is typically $10^9 - 10^{10}$ Pa. Hence

720
$$\sigma_{22}^{hs} = M W_z^{hs} = \rho_i g (z+d) + S_{bt}^{hs} \in [S_{bt}^{hs}, S_{bt}^{hs} + \rho_i g h], \qquad (A 8a)$$

721
722 and
$$M W^{\text{hs}} = \frac{1}{2} \rho_{\text{i}} g (z+d)^2 + S^{\text{hs}}_{\text{bt}} (z+d),$$
 (A 8b)

where S_{bt}^{hs} is the unknown stress at the bottom of the ice. W^{hs} satisfies (A 6d), while (A 6c) is satisfied if

725
$$-\frac{S_{\text{bt}}^{\text{ns}}}{M} = \frac{2P_{\text{at}} + \rho_{\text{i}} g h (2M + \rho_{\text{i}} g h)}{2M (M + \rho_{\text{i}} g h)} = \max\left\{ \left| W_{z}^{\text{hs}} \right| \text{ for } z \in [-d, h - d] \right\}.$$
(A9)

Since the atmospheric pressure $P_{at} \approx 10^6$ Pa, $|W_z^{hs}| \ll 1$ if the ice thickness $h \ll M/(\rho_i g) \approx 5 \times 10^5$ m or 500 km. Hence, for typical ice shelves, gravitational compression should have negligible effect on wave propagation.

729 Appendix B. Static version of thin plate equations ($\omega = 0$)

The static version ($\omega = 0$) of the thin plate equations (5.12a,c) are, respectively,

731
$$F w''' + w = 0$$
 and $u'' = 0$ for $x > 0$. (B 1a,b)

The corresponding static versions of the shelf front conditions (5.26a-c) are

733
$$Fw'' - p_1 = 0$$
, $w''' = 0$ and $Gu' + p_0 = 0$ for $x = 0$, (B 2a,b,c)

734 where

735
$$p_0 = \frac{1}{\rho_w g} \left\{ \int_{-d}^0 (P_{at} + \rho_w g z) \, dz + \int_0^{h-d} P_{at} \, dz \right\}$$

736

$$=\frac{P_{\rm at}h}{\rho_{\rm w}g}-\frac{d^2}{2},\tag{B3}$$

737 and
$$p_1 = \frac{1}{\rho_w g} \left\{ \int_{-d}^0 \left(d - \frac{h}{2} + z \right) \left(P_{at} + \rho_w g z \right) dz + \int_0^{h-d} \left(d - \frac{h}{2} + z \right) P_{at} dz \right\}$$

$$= -\frac{d^2}{2} \left(\frac{d}{3} - \frac{h}{2}\right). \tag{B4}$$

738 739

The static flexure, $w = w_{st}$, satisfying Eqs. (B 1a), has bounded solutions of the form

741
742
$$w_{\rm st}(x) = e^{-\beta x} \left(C_+ e^{i\beta x} + C_- e^{-i\beta x} \right), \tag{B5}$$

where $\beta = (4 F)^{-1/4}$ and C_{\pm} are determined from (B 2a,b). The general solution for the static extension, $u = u_{st}$, satisfying Eqs. (B 1b) is

$$u_{\rm st}(x) = A x + B. \tag{B6}$$

746 However, it has no bounded solutions satisfying the shelf front condition (B 2c).

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